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# STUDY OF HYDRAULIC FILTER LEVEL OFFSET (HyFLO) EQUIPMENT FOR AUTOMATIC DOWNSTREAM CONTROL OF CANALS

J. C. Schuster E. A. Serfozo Engineering and Research Center Bureau of Reclamation

January 1972



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by<sup>3</sup> J. C. Schuster E. A. Serfozo

# January 1972

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Division of General Research Division of Design Engineering and Research Center Denver, Colorado

UNITED STATES DEPARTMENT OF THE INTERIOR Rogers C. B. Morton Secretary

BUREAU OF RECLAMATION Ellis L. Armstrong Commissioner

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Laboratory simulation of mathematical model prediction
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Corning Canal (pressure transducer sensor)
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# INTRODUCTION

A development program with the University of California at Berkeley, sponsored by the Bureau of Reclamation, produced a mathematical model and equipment for automatic downstream control of canal check gates.<sup>1</sup> <sup>2</sup> <sup>3</sup>\* The feedback control system, named Hydraulic Filter Level Offset (HyFLO), automatically adjusts the canal inflow from water level offsets caused by the canal outflow (Figure 1). Changes in the position of the upstream gate are proportional to the offsets in water level, Figure 2.





The HyFLO method is a closed-loop control system in which the control action is proportional to the downstream water level. The response of the controller has to be in "real time" which is governed by the time constant of the canal section under control. The time response of the system is in the order of 2,000 to 3,000 seconds. Because of the long time constants, a hydraulic time delay circuit was designed for the HyFLO control system. The hydraulic filter includes a capillary damping tube open to the canal stilling well level and connected to a smaller secondary well. Flow into and out of the secondary well is computed to have a linear relation to the head across the capillary. A sufficiently long time constant necessary to control the canal flow was obtained easily with the hydraulic delay circuit. Field trial of the method was desirable to confirm the analytical studies.

\*Numbers refer to References at end of text.





Equipment was built to verify the mathematical model and the Corning Canal, Central Valley Project, California, was used for the test installation. A series of conditions were simulated on a digital computer using the mathematical model to predict the water level changes and gate movement for specified outflows. These outflows were repeated in the canal with the HyFLO equipment installed to control four radial check gates. During the test period, difficulties were experienced in the electronic circuitry so the test results were not entirely satisfactory. Since difficulties in the electronics were experienced during the field tests, the transmitters, receivers, and the analog computer-controller were sent to the Engineering and Research Center, Hydraulics Branch, Laboratory assistance was requested in resolving a transducer sensor malfunction, a power supply failure, and "noise" in the receiving and transmitting system. A laboratory simulation of a canal reach was constructed through the efforts of Electrical, Hydraulic Structures, and Hydraulics Branches.

# ANALOG COMPUTER-CONTROLLER

# General

The computer-controller includes operational amplifiers for analog computing of the appropriate gate position, Figure 3. The input signal to the controller is the delayed signal output of the float-driven potentiometer in the hydraulic filter. The hydraulic filter must be located near the first check downstream from the controlled check. The voltage change of the float-driven potentiometer is converted to a variable 5to 25-hertz signal, utilizing a voltage-to-frequency analog transmitter. The low-frequency signal is transmitted to an appropriate receiver at the controlled check structure.

The receiver converts the 5- to 25 hertz signal to a current output of 0 to 5 milliamperes. The output current is used to operate the analog computer-controller. Operation of the controller is as follows:

The initial conditions for the analog computer are entered via the operational amplifier feedback resistors and input potentiometer adjustment. A "target depth" adjustment must be made to define the "zero flow", starting point for the controller. The actual gate position is fed into the analog-computer to provide the necessary feedback for "anti-hunt" operation. The gate position is converted to a voltage suitable for operation of the analog computer by a potentiometer connected to the gate drive motor shaft.

The output of the analog computer operates into complementary Schmitt triggers. The Schmitt triggers operate when the positive or negative output voltage of the analog computer reaches a preset value. When the preset value is reached, the appropriate "Raise" or "Lower" Schmitt trigger operates and energizes the gate motor. The gate motor will run until the value of the analog computer output reaches an increased or decreased preset value. The output signal of the analog-computer is regulated from the feedback signal input of the gate position potentiometer. The gate will Sperate for a fixed time determined by the voltage "'on" and "off" adjustments preset into the Schmitt triggers. The time between gate operations is proportional to the rate-of-change of the water level at the downstream gaging point. The "hydraulic filter" time constant provides the proper time for gate operations.

## Modifications to Original Equipment

The operation of the analog computer-controller was improved by modifying the original equipment as follows:

 $\mathbb{C}$ 



Figure 3. Analog computer controller diagram.

a. Chassis and circuit commons were separated.

b. Linearity of the Schmitt triggers was improved by changing component values. Stability was improved with addition of capacitors.

c. An impedance matching amplifier was inserted between the gate potentiometer output and the analog computer input to prevent loading.

d. A 100-µf capacitor was inserted across the output of the Quindar QATR-20 receiver to improve its low-frequency performance while operating into operational-amplifier-type circuits.

e. A 0.001- $\mu$ f capacitor was installed across resistor R 27 in the QATR-20 receiver to prevent high-frequency self-oscillation due to operation into the first operational amplifier of the analog computer-controller.

f. RC networks were added across the coils of the transmitting and receiving relays to eliminate

high-voltage spikes being produced by switching of the high-impedance coils.

g. Shielded wire was used to connect all inputs to analog computer-controller to reduce 60-hertz pickup.

# LABORATORY SIMULATION

Equipment was installed in the Hydraulics Branch to simulate one section of canal between two radial gates (heavy dashed line), Figure 4. A steel tank representing the canal stilling well was connected to a metered water inflow and drain to change the water level manually. The hydraulic filter well or time delay circuit was installed in the well. The analog computer-controller, transmitter-receiver, and simulated gate hoist were placed on an adjacent table, Figure 5.

The voltage output necessary to operate the transmitter was developed in the hydraulic time delay





Photo PX-D-70737

Figure 5. Laboratory simulation of canal section.

circuit using a float-operated potentiometer. The movement of a 2-1/2-inch (6.3-cm) diameter float in the delay well was transferred by a perforated brass tape to 1 2-inch (5.1-cm) diameter pulley on the potention eter shaft, Figure 6a. The voltage change in the potentiometer is a measure of the offset of the water level from a preset "target" level, Figure 1. The time delay between the change in the canal stilling well and the hydraulic filter well level was controlled by the ratio of the inside diameters of the filter well 4 inches (10.2 cm) and capillary tube, 0.13 inch (3.3 mm), and the 80-inch (2-m) length of capillary tube. Mathematical relationships given in the analysis were used to size the tube and well. The response of the system was limited by laminar flow in the capillary tube to minimize the potential oscillation of a wave traveling between two gate structures, Figure 4.

# **OPERATION**

The main objectives of the simulation in the laboratory were to improve the performance and the reliability of the field equipment. Because difficulties had been encountered in the field, the laboratory problem included the developing and refining of the hydraulic



a. Float pulley on shaft of potentiometer, and perforated tape attached to float. Photo PX-D-70732

Û

b. Assembled components. Photo PX-D-70730

c. Stilling well parts except 2-1/2-inch float in well. Photo PX-D-70731

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 $\odot$ 

d. Capillary damping tube and bypass drain. Photo PX-D-70733

 $\langle \rangle$ 

Figure 6. Components of hydraulic time delay and water level sensor.  $\widehat{\mathbb{N}_{2}}$ 

5

 $^{\circ}$ 

time delay circuit and water level sensor and improving the operation of the electronic controller. The input to the laboratory system corresponded to the input to the mathematical model. A water level change in the canal stilling well caused by a 50-cfs (1.4-cu m/sec) delivery from the section was computed from the mathematical model, Figure 7. The indicated change was manually controlled into and out of the laboratory stilling well. Records were made of the change in stilling well level, the water level in the filter well, and the indicated gate opening. The laboratory records were then compared to the predictions from the mathematical model.

# CONTROL SYSTEM STUDY RESULTS

## **Hydraulic Filter**

A continuous change in the output of the potentiometer with changes in the water surface level in the filter well is a desirable characteristic. Initial operation of the control system showed the 2-1/2-inch (6.3-cm) diameter float produced too small a force to smoothly drive the tape, pulley, and potentiometer. In the attempt to minimize the filter well size (4 inches, 10.2 cm) for ease of installation, the float was too



Figure 7. Transients predicted by mathematical model of Corning Canal.

small to overcome static friction. This resulted in not being able to repeat the target elevation.

A 0.15-foot (45.7-mm) step in gate opening was selected to prevent continuous operation of the hoist motors of the radial gates. The design of the control system provided this 0.15-foot step for a water level offset of 0.019-foot (5.7-mm). A water level change of 0.019 foot on the 2-1/2-inch-diameter float produced a force of about 0.6 ounce (17 grams). A 2-inch (5.1-cm) diameter pulley applied 0.6 ounce inches (43.2 gr-cm) to the potentiometer. Measurements of torque showed an 0.18-ounce (5-gram) weight at 1 inch was sufficient to turn the potentiometer shaft. The static friction became too large for reliable movements after the float, tape, and counterweight were added to the pulley, Figure 6a.

A study was then made to find a float size or a substitute means for measuring the water level with greater sensitivity. Because of the satisfactory resolution and sensitivity of a pressure transducer, a sensor was constructed to replace the float. A 1-psi strain-gage-type transducer was connected to a 1/8-inch (3.1-mm) outside diameter by 3/32-inch (2.4-mm) inside diameter brass tube. The tube and transducer pressure chamber were filled with distilled water. The tube was inserted into the filter well in place of the tape and float; Figure 8. For a computed time constant of 1,900 seconds in the laboratory hydraulic filter, the gate responded seven times in opening and two times in closing during operation, Figure 9.



Figure 8. Water level sensing using water filled brass tube and pressure transducer, Photo PX-D-70735

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Reversal of the time scale in the laboratory data was necessary because the maximum water surface elevation was also the maximum voltage of the system. Time references were placed at the right instead of the left of the recorder chart, the reverse of Figure 7.

The pressure transducer was an ideal water level sensor but required an additional high-stability signal conditioning circuit. Simplification of the system suggested that the float-potentiometer sensor be used but increased sensitivity be obtained by a larger float. The uniformity of the output of the potentiometer was measured for 5- and 12-inch (12.7- and 30.5-cm) diameter floats attached to the tape, Figure 10.

The 12-inch float, tape, potentiometer, and counterweight reacted to water level changes in the stilling well with a sensitivity equal to the pressure transducer. To accommodate, the 12-inch float an unwieldy size of filter well would be necessary and cause difficulty in installation.

Because of the smaller size and good response a 5-inch float was selected for investigation and a 6-inch (15.3-cm) filter well was constructed to accommodate the float. Operation of the system using a computed time constant of 1,500 seconds produced seven steps in opening the gate and three steps in closing the gate, Figure 11.

The time of the gate opening was controlled by the sensitivity of adjusting the voltage of the Schmitt trigger. Variance in the order of plus or minus 0.03 volt in 1 volt of level was part of the cause of the shift shown by the time marks from the mathematical model placed near the recorded gate openings on Figures 9 and 11.

The potentiometer voltage output was not as uniform as that for the pressure transducer. Friction again caused a slight stepwise variation but the voltage steps were small. The reaction of the control appeared to be unaffected by the slight nonuniformity of potentiometer output.

Agreement between the mathematical model and laboratory simulation was better with the float than with the transducer in their respective stilling wells. The mathematical model computed the transients in Figure 6 from a time constant of 1,900 seconds. The filter well containing the transducer had a computed time constant of 1,900 seconds and a measured constant of 2,660 seconds for a 0.22-foot step function





in head. The filter well for the float had a computed time constant of 1,500 seconds and a measured constant of about 2,150 seconds for a 0.21-foot step function in head. The time constant of 2,150 seconds was closer to the mathematical model and thus the hydraulic filter with float gave better correspondence between theory and experiment.

# **Control Time Constant**

Analysis. – Wave travel times in the sections of Corning Canal had been computed by personnel at the University of California.<sup>3</sup> These times were used with derived equations to compute the sizes of the stilling well and capillary damping tube, Figures 6c. and d. The equation of continuity for the filter is:

q<sub>f</sub> = 
$$\frac{ad∆h_{f}}{dt}$$

(1)

٩f	=	the flow into the filter well
a	=	the water surface area in the well
∆h <sub>f</sub>	=	the change in the water level in the well
dt	. =	change in time corresponding to the well change

The energy equation for head loss across the capillary with entrance and exit losses neglected is:

$$\Delta y - \Delta h_{f} = \frac{fL}{2gd_{c}} \frac{qf}{\pi d_{c}^{2}/4}^{2}$$
(2)  
ere  
$$\Delta y = \text{ the displacement in canal depun} from a steady statef = the friction factorL = length of the capillary tubedc = the inside diameter of the tube$$

For linear damping, flow through the capillary tube should be laminar. The friction factor is thus related to the Reynolds Number by:

$$f = \frac{64}{N_R}$$
(3)  

$$e = \frac{(q_f/\pi d_c^2)}{4} \frac{d_c}{\nu}$$
(4)  

$$= \text{ the kinematic viscosity} \\ \text{ of water}}$$

Equations (1), (2), (3), and (4) combine to give a differential equation for the hydraulic filter:

$$\frac{\Delta hf}{dt} + \frac{1}{ak} \left( \Delta hf - \Delta \gamma \right) = 0$$

(5)

where

wher

v

where

whe

$$k = \frac{128\nu L}{\pi g d_c^4}$$

The initial conditions of control at time t equal to 0 will have the canal level and filter well level at the same elevation. Thus  $\Delta hf$  will be equal to 0 and the solution to equation (5) is:

9



$$\Delta hf = \Delta y (1 - e^{-t}/t_f)$$

(6)

where

$$t_f = ak = time constant for the hydraulicfilter $\Delta y = canal level change$$$

The time constant  $\tau$  in terms of the physical dimensions of the filter is:

$$\tau = ak = \frac{(\pi d^2 f)}{4} \frac{(128\nu L)}{\pi g d_c^4}$$
(7)  
$$\tau = ak = \frac{32\nu L}{2} \frac{d f^2}{d_c^4}$$

*Time constant measurements.*—The time constant of the hydraulic filter was measured by monitoring the voltage output of the float-driven potentiometer and noting the time elapsed for the initial voltage to decrease to 63 percent of its value. The value of 63 percent of the maximum voltage is based on the following equation for the discharge of a capacitor:

Vmax = Maximum voltage output of potentiometer at t = 0

where

$$T = time t_f = filter time$$

for:

 $T = t_{f}$  the equation reduces to:

$$T = V_{max} (1 - e^{-1}) = V_{max} \left(1 - \frac{1}{e}\right)$$
$$T = V_{max} (1 - 0.3679) = 0.6321 V_{max}$$

Major differences were found between the times for the filter to reach 0.63 of a step function as computed and measured in the laboratory. The indicated time constant varied with the size of the step function, Figure 12.

For the capillary tube (0.9-inch (22.8-mm) long by 1/32-inch (0.8-mm) inside diameter), time constants for small step functions were in closer agreement to the computed value. Computations of Reynolds Numbers from measured data showed that near turbulent flow conditions were present for the larger steps, Thus,

conventional analysis does not appear to account for the excess entrance and turbulent losses occurring in the tube. Time constants associated with small steps or ramp variations normally encountered in the level change in the canal stilling well would be more closely computed by Equation 7.

Table 1 is a summary of the computed and measured times for a series of capillary tubes used in this study. The time constants apply to both the 4- and 6-inch filter wells. Data were taken throughout the study of the HyFLO control as filter well configurations were changed to provide desired time constants.

Fears were expressed that debris from the canal water passing from the stilling well to the filter would plug the capillary tubing. Because of the probability, studies of the HyFLO control system included a plastic bag submerged in the stilling well to supply clean water to the filter, Figure 13. Time constants in Table 1 indicate that no appreciable change was caused by the addition of plastic bag and tubing.

Stilling well measurements.—Use of the canal stilling well and a larger float (12 inches or more) was suggested as a possible substitute for the control hydraulic filter. Limited studies were made of the time constants of the laboratory well, Table 2.

For the three tubes cited, Reynolds Numbers near the transition range of 2,000 occurred for step  $\Delta H$  values of 0.2 foot. Excess losses were evidenced by the lack of agreement between the computed and measured time constants.

# CONCLUSIONS

Completion of the laboratory studies showed:

1. The HyFLO system equipment with slight modifications was capable of automatically controlling a gate supplying water to a canal section. Modifications included improving electrical components of the computer-controller and increasing the size of the hydraulic filter to accomodate a 5-inch float in place of a 2-inch float.

2. The control would supply water on demand from a delivery assumed to be located near the downstream end of a canal section.

3. A hydraulic time delay circuit can provide a delay of sufficient length to minimize oscillations when changes are made in canal flows.



Figure 12. Time constant measurements HyFLO filter with pressure transducer sensor.

 $\overline{a}$ 

4. The laboratory model satisfactorily simulated predictions of gate opening and water level changes in the canal section. Exact correspondence was not obtained between the mathematical model predictions and the laboratory simulation because of the voltage sensitivity of the Schmitt triggers in the controlier and actual filter time constants larger than computed from equations in the mathematical model.

5. Additional studies of the hydraulics of the time delay circuit should be made to produce an analysis to account for the loss across the capillary tube in excess of that predicted by equations from the mathematical model.

6. Additional analytical and experimental studies should be made of the system time constant to determine the range of variance for satisfactory flow control.

7. The addition of a plastic bag containing clear water will assist in keeping the capillary tube clean and the hydraulic time delay operating. The time constant of the delay was not appreciably changed by the addition of the bag and connecting tubing.

8. Substitution of a stable electronic time delay would allow use of a large float in the stilling well and would eliminate the maintenance necessary to assure satisfactory operation of the hydraulic time delay.

9. A float, tape, pulley; and potentiometer combination is a simple and satisfactory way of sensing water level and transferring information to the analog-computer. A 5-inch float produced a satisfactory force to operate the sensor. A larger float would be desirable for long-term operation of the mechanical parts as friction increases with age,

# Table 1

Capillary tube		Computed	2 	Measured*		
Inside	Length	time	time Step time			
diameter	(inches)	constant	ΔH	constant	Remarks	
(inches)		(seconds)	(feet)	(seconds)		<u></u>
			(10.2-cm) Dia	meter Stilling Well		
			pressure trans	ducer sensor)		
1/32	0.9	1,900	1.0	3.940	Flow out of filter well	water
(0.8 mm)	(22.8 mm)	.,		0,010	temperature	70 <sup>0</sup> F
I/d = 28.8	(22.0)		0.22	2 660		66 <sup>0</sup> F
			0.10	2,285	9	66 <sup>0</sup> F
			<i>N</i>			
0.13	80	490	1.0	870	Out. Temp	66 <sup>0</sup> F
(3.3 mm)	(2 m)			0.0		
L/d = 625	1,		10	930	-	59 <sup>0</sup> F
_,						
			2-1/2-inch F	loat Sensor		
1/32	0.9 🗢	1.900	0.18	5.400	Out. Temp	72 <sup>0</sup> F
	V.		0.11	5,280		72 <sup>0</sup> F
			0.2	**4.650	In. Temp	72 <sup>0</sup> F
		19 1	0.11	5,350		72 <sup>0</sup> F
				-,		- · ·
	1	6-inch	(15.3-cm) Dia	meter Stilling Well		
			(5-inch flo	at sensor)		
	5.75					14
3/32	28.8 💛	1,500	0.22	2,180	Out, Temp	78 <sup>0</sup> F
(2.4 mm)	(73 cm)			_,		
L/d = 307		. `	0.20	1,980	In, Temp	78 <sup>0</sup> F
	]					
	5-inch	(12.7-cm) Float,	Plastic Bag, ar	nd 12 feet of 3/8-in	ch Plastic Tubing	
		1,500	0.21	2,190	Out, Temp	70° F
	1 .		0.21	I 2,120 ⊟	In Temp	70° F

# SUMMARY OF FILTER WELL TIME CONSTANTS

\*Time required to reach 0.63 of  $\Delta H$ .

••Cause not determined.

10. A pressure tranducer has greater sensitivity to water level changes, but stable signal conditioning equipment and transducer are required for continued satisfactory operation.

11. Before general use is attempted, of the HyFLO system, a careful study should be completed of the operating characteristics of a single canal controlled by multiple units.

# APPLICATIONS

Upon conclusion of these laboratory studies, the filter and a modified comparator were returned to California and installed to control the Corning Canal section simulated in the laboratory. An initial trial period of about 6 months prior to this report resulted in satisfactory control of flow in the section.



Figure 13. Hydraufic time-delay 6-inch stilling well and plastic bag for clean water supply. Photo PX-D-70738

Refinement of the HyFLO downstream control equipment and method is possible. Further study of the hydraulics of the filter should be made to produce an analysis to account for the excess loss in the capillary damping system. A study of a stable electronic time delay could result in the elimination of the hydraulic time delay. The electronic delay would allow use of a large float and potentiometer mechanism to increase reliability and reduce maintenance costs of the control equipment. Time constant studies in an operating canal should include the allowable variance to maintain satisfactory control of the flow.

The HyFLO method of downstream control and equipment of increased reliability should have general use in existing and proposed water systems of the Bureau. Before general use is attempted, a careful study should be completed of the operating characteristics of a single canal controlled by multiple units of HyFLO equipment.

# REFERENCES

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2. Shand, M. J., "Final Report—The Hydraulic Filter Offset Method for the Feedback Control of Canal Checks," Report No. HEL-8-3, Hydraulic Engineering Laboratory, College of Engineering, University of California, Berkeley, June 1968.

3. Shand, M. J., Thesis, "Automatic Downstream Control System for Irrigation Canals"-Unpublished.

# Table 2

Tube size		Computed		Measured*	
Inside diameter (inches)	Length (inches)	time Sconstant (seconds)	Step ∆H (feet)	time constant (seconds)	Remarks
0.44 (11.2 mm)	600 (1,525 cm)	2,440	1.0	7,440	Copper tubing T = 70 <sup>0</sup> F
			0.2	4,570	T = 71 <sup>0</sup> F
0.44	504	2,060	1.0	5,660	Saran plastic
(11.2 mm)	(1,281 cm)		0.5	4,760	T = 69 <sup>0</sup> F
÷ .	· .		0.2	3,800	
0.88 (22.4 mm)	540 (1,372 cm)	_	0.2	700	Garden hose T = 70 <sup>0</sup> F

# SUMMARY OF STILLING WELL TIME CONSTANTS

\*Time required to reach 0.63 of  $\triangle$ H. Stilling well diameter 34.5 inches (87.7 cm).

. Ø @

Acre-feet

## CONVERSION FACTORS-BRITISH TO METRIC UNITS OF MEASUREMENT

The following conversion factors adopted by the Bureau of Reclamation are those published by the American Society for Testing and Materials (ASTM Metric Practice Guide, E 380-68) except that additional factors (\*) commonly used in the Bureau have been added. Further discussion of definitions of quantities and units is given in the ASTM Metric Practice Guide.

The metric units and conversion factors adopted by the ASTM are based on the "International System of Units" (designated SI for Systeme International d'Unites), fixed by the International Committee for Weights and Measures; this system is also known as the Giorgi or MKSA (meter-kilogram (mass)-second-ampere) system. This system has been adopted by the International Organization for Standardization in ISO Recommendation R-31.

The metric technical unit of force is the kilogram-force; this is the force which, when applied to a body having a mass of 1 kg, gives it an acceleration of 9.80665 m/sec/sec, the standard acceleration of free fall toward the earth's center for sea level at 45 deg latitude. The metric unit of force in SI units is the newton (N), which is defined as that force which, when applied to a body having a mass of 1 kg, gives it an acceleration of 1 m/sec/sec. These units must be distinguished from the (inconstant) local weight of a body having a mass of 1 kg, that is, the weight of a body is that force with which a body is attracted to the earth and is equal to the mass of a body multiplied by the acceleration due to gravity. However, because it is general practice to use "pound" rather than the technically correct term "pound-force," the term "kilogram" (or derived mass unit) has been used in this guide instead of "kilogram-force" in expressing the conversion factors for forces. The newton unit of force will find increasing use, and is essential in SI units.

Where approximate or nominal English units are used to express a value or range of values, the converted metric units in parentheses are also approximate or nominal. Where precise English units are used, the converted metric units are expressed as equally significant values.

## Table 1

QUANTITIES AND UNITS OF SPACE To obtain Multiply Вγ LENGTH Mil : 25.4 (exactly) Micron Millimeters Inches 25.4 (exactly) 2.54 (exactly) Inches Centimeters Feet 30.48 (exactly) Centimeters 0.3048 (exactly) Meters Feet ....... Feet ..... 0.0003048 (exactly) Kilometers Yards Meters Miles (statute) Meters 1.609344 (exactly) Kilometers AREA Square inches . . . . . . . . . . . 6.4516 (exactly) ..... Square centimeters Square feet 0.092903 Square feet Square yards ..... 0.836127 ..... Square meters ..... Hectares 0.40469 . . . . . Square meters 4.046.9 0.0040469 . . . . . . . . . . . . . Square kilometers Acres . . . . . . . . . . . . . . . . 2.58999 ..... Square kilometers Square miles ...... VOLUME Cubic inches . 16.3871 Cubic centimeters Cubic feet 0.0283168 Cubic meters 0.764555 ..... Cubic meters Cubic yards . . . . . . . . . . . . . CAPACITY Fluid ounces (U.S.) 29.5737 . . Cubic centimeters Fluid ounces (U.S.) Mittiliters Liquid pints (U.S.) 0.473179 ..... Cubic decimeters .... Liters ..... Cubic centimeters Quarts (U.S.) ..... 946.358 0.946331 ..... Quarts (U.S.) .... Liters 3,785.43 . . . . . . . . . . . . . . . . Cubic centimeters Gallons (U.S.) . . . . . . . . . 3,78543 ..... Cubic decimeters Gallons (U.S.) . . . . . . . . . . 3,78533 Gallons (U.S.) . . . . . . . . Galtons (U.S.) . . . . . . . . \*0.00378543 . . . . . . . . . . . . . . . . . . . Cubic meters Gallons (U,K,) 4.54609 ..... Cubic decimeters 4.54596 L iters Gallons (U.K.) Cubic feet 28.3160 . Liters Cubic yards 764.55 Liters Acre-feet

## Table II

	ANTITIES AND UNITS OF ME	HANICS
Maloply	By	To obtain
	MASS	
Grains (1/7,000 lb) Troy ounces (4809 grains) Ounces (avdp) Pounds (avdp) Short tons (2,000 lb) Short tons (2,000 lb) Long tons (2,240 lb)	64.79891 (exactly) 31,1035 28,3495 0.45359237 (exactly) 907.185 0.907185 1,016.05	Milligram Gram Gram Kilogram Kilogram Metric ton Kilogram
	FORCE/AREA	
Pounds per square inch Pounds per square inch Pounds per square foot Pounds per square foot	0.070307 0.689476 4.86243 47.8803	Kilograms per square centimete     Newtons per square centimete     Kilograms per square mete     Newtons per square mete
	MASS/VOLUME (DENSITY)	
Ounces per cubic inch Pounds per cubic foot Pounds per cubic foot	1.72999 16.0185 0.0160185 1.32894	Grams per cubic centimeter Kilograms per cubic meter Grams per cubic centimeter Grams per cubic centimeter
	MASS/CAPACITY	
Ounces per gallon (U.S.) Ounces per gallon (U.K.) Pounds per gallon (U.S.) Pounds per gallon (U.K.)	7.4893 6.2362 119.829 99.779	Grams per liter Grams per liter Grams per liter Grams per liter Grams per liter
	BENDING MOMENT OR TOR	DUE
Inch-pounds Inch-pounds Foot-pounds Foot-pounds per inch Foot-pounds per inch Ounce-inches	0.011521 1.12985 × 10 <sup>6</sup> 0.138255 1.35582 × 10 <sup>7</sup> 5.4431 72.008	Meter-kilograms Centimeter-dynes Meter-kilograms Centimeter-dynes Centimeter-kilograms per centimeter Gram-centimeter
	VELOCITY	
Feet per second Feet per second Feet per year , Miles per hour Miles per hour	30,48 (exactly) 0.3D48 (exactly) *0.965873 x 10 <sup>-6</sup> 1.609344 (exactly) 0.44704 (exactly)	Centimeters per second Meters per second Centimeters per second Kilometers per hour Meters per second
	ACCELERATION*	
Feet per second <sup>2</sup>	*0.3048	Meters per second <sup>2</sup>
	FLOW	
Cubic feet per second (second-feet) Cubic feet per minute Gallons (U.S.) per minute	*0.028317 0.4719 0.06309 FORCE*	
Pounds Pounds Pounds	*0.453592 *4.4482 *4.4482 × 10 <sup>5</sup>	Kilograms Newtons

35

•

#### Table II-Continued

Multiply	By	To obtain
	JRK AND ENERGY"	
itish thermal units (Btu)	10 252	#11
tisk thermal units (Btu)		Kilogram calories
		Joules
perpound ,	2.326 (exactly)	Joules per gram
	1.35582	Joule:
	POWER	
rrapower 7	45 700	
per hour	0 202071	Watts
pt-pounds per second	1.35582	Watt
	UPAT TOANGEED	
	HEAT THANSFER	
in ,/hr ft <sup>2</sup> degree F (k,	÷ .	
hermal conductivity)	1.442	Milliwatts/cm degree C
u in /hr ft= degree F (k,		
rermal conductivity)	0.1240	Kg cal/hr m degree C
tt/hr ft- degree F	1.4880	Kg cal m/hr m≤ degree (
/hr ft² degree F (C		
termal conductance	0.568	Milliwatts/cm* degree C
armal conductance)	4 000	N Ka cal/ba -2 damas (
rao E br ft2/Rtu /R	4,002	Ng cai/nr m- begree C
per ( ( it ) Did ( it )	1 761	Derree C am2/millium
//////////////////////////////////////	A 1968	Va dograe (
/lb degree F	*1.000	Callorer degree (
/hr [thermal diffurivity]	0 2591	
(hr (thormal diffusivity)	*0.00200	M2/2
/in futer that officiality	0.03230	
WA	TER VAPOR TRANSMISS	ION
ains/hr ft= (water vapor)	10.7	
	0,5	Matria and
minahar (narmatilitu)	1.67	hforrin ports continutor
in the permeasurity		···· end permocentimeters
	· · · ·	
· .		
	- Table III	
OTH	IFR QUANTITIES AND U	NITS
Multiply	Ву	To obtain
Dic feet per square foot per day (seenage	*304.8	Liters per gruppe meter per de
ind seconds per square foot (viscosity)	*4 8924	Kiloman second per square meter per day
are feet per second (viscosity)	10 092903	Shuare meters our second
renbeit decrees (chapge)	5/9 gyactly	Celsius or Keluin degrees (chapped)
ts per mil	0 03027	Kilouoita par willingen
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## ABSTRACT

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DESCRIPTORS—/ downstream/ \*control/ automation/ \*automatic control/ irrigation canals/ \*canals/ hydraulics/ damping/ timing circuits/ control systems/ water levels/ water level fluctuations/ offsets/ analog computers/ laboratory tests/ test results/ simulation/ mathematical models

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